

## MOLAR SPECIFIC HEAT AT CONSTANT VOLUME

THE INITIAL STATE OF THE GAS IS HEATED TO FINAL STATE

$$\text{HEAT} = Q$$

$$Q = n C_v \Delta T$$

CHANGE IN INTERNAL ENERGY

$$\Delta E_{\text{int}} = n C_v \Delta T - W$$

$W = \text{WORK DONE}$

$$\text{IF } W = 0$$

$$\Delta E_{\text{int}} = n C_v \Delta T$$

$$C_v = \frac{\Delta E_{\text{int}}}{n \Delta T}$$

CHANGE

$$\Delta S = S_f$$

HEAT LOSS

$$\Delta S_{\text{gas}}$$

HEAT GAIN

$$\Delta S_g$$

FORCE



TO FINAL STATE

CHANGE IN ENTROPY

$$\Delta S = S_f - S_i = \int_i^f \frac{dQ}{T}$$

HEAT LOSS

$$\Delta S_{\text{gas}} = - \frac{|Q|}{T}$$

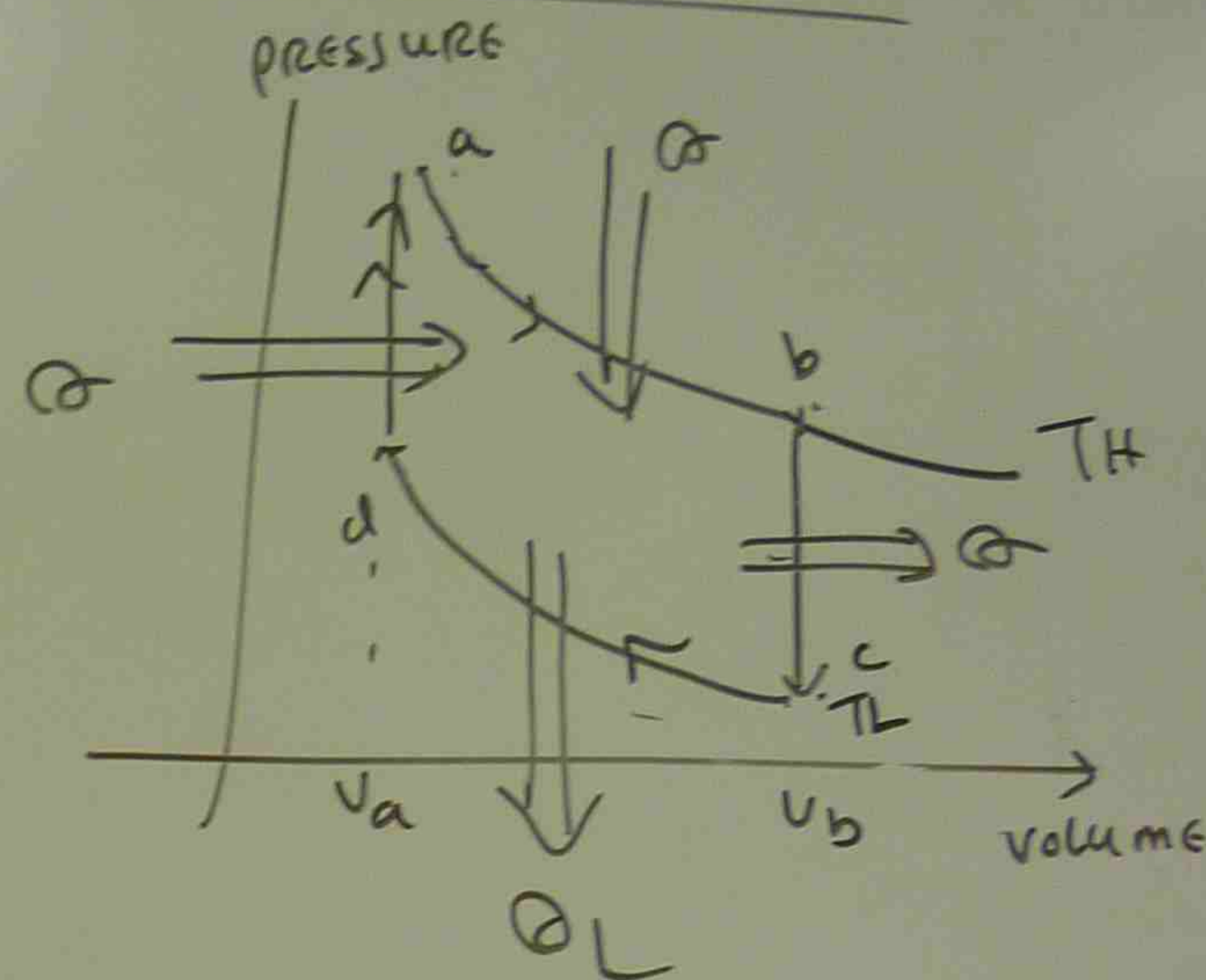
HEAT GAIN

$$\Delta S_{\text{gas}} = + \frac{|Q|}{T}$$

FORCE DUE TO ENTROPY

$$dE = dQ - dW$$

STIRLING ENGINE



CARNOT CYCLE

Compress the gas c → d

Volume  $V_b \rightarrow V_a$

Heat  $Q_L$  dissipates

Heat the gas at d → a

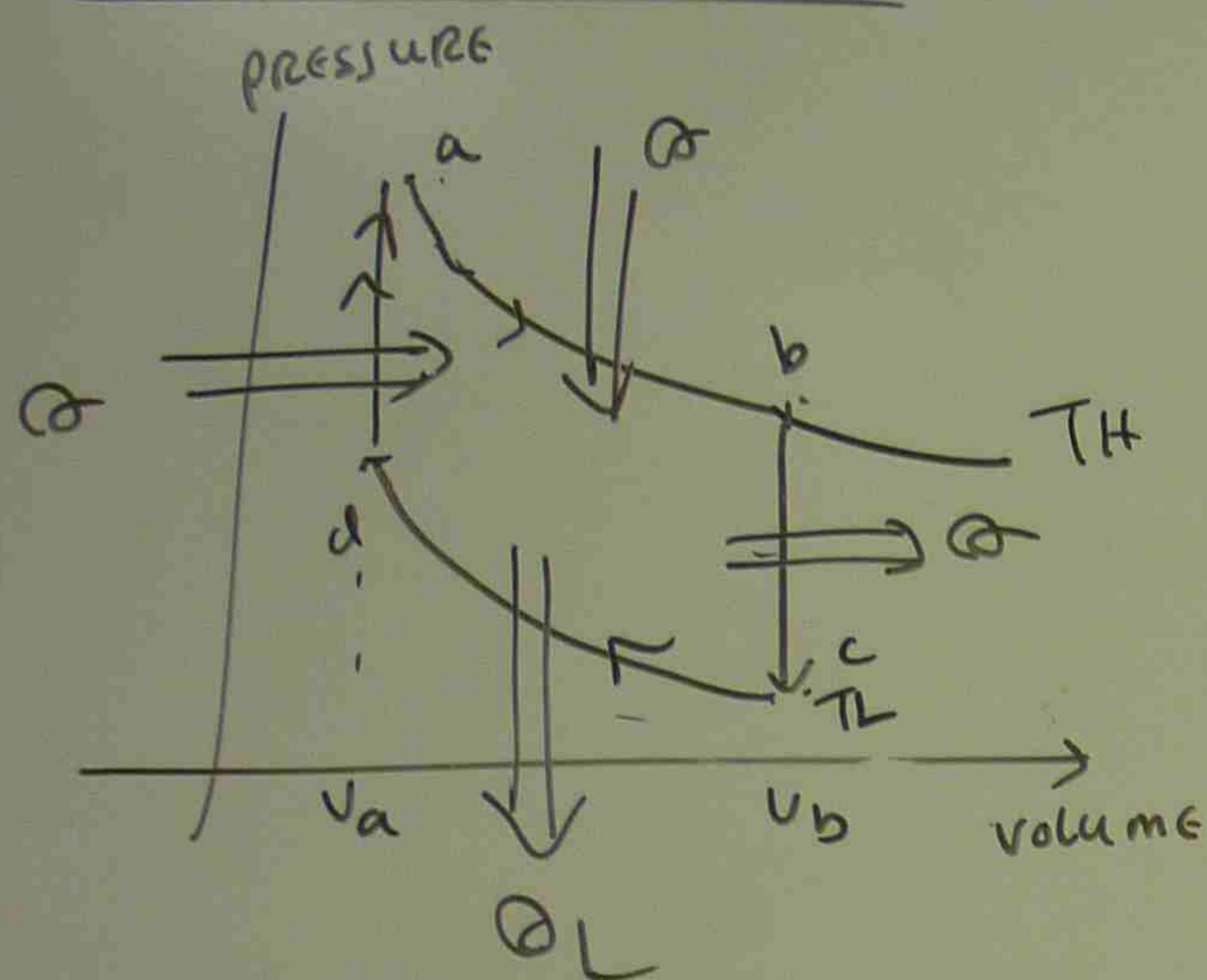
Temperature rises from  $T_L$  to  $T_H$

Pressure increases

Heat absorbed.



## STIRLING ENGINE



CAR NOT CYCLE

Compress the gas  $c \rightarrow d$   
volume  $V_b \rightarrow V_a$

HEAT  $Q_L$  DISSIPATES

HEAT THE GAS AT  $d \rightarrow a$

TEMPERATURE RISES FROM  $T_L$  TO  $T_H$

PRESSURE INCREASES

HEAT ABSORBED.

EXPANSION

$a \rightarrow b$   
VOLUME  $V_a \rightarrow V_b$

COOLING

$b \rightarrow c$

HEAT DISSIPATED  
TEMPERATURE DECREASES  
FROM  $T_H$  TO  $T_L$

pb THREE CARNOT ENGINES OPERATE BETWEEN  
RESERVOIR TEMPERATURES OF (a) 400K AND 800K  
(b) 600 AND 800K (c) 400 AND 600K. RANK  
THE ENGINES ACCORDING TO THEIR THERMAL  
EFFICIENCIES. GREATEST FIRST.

$$\text{Efficiency} = 1 - \frac{T_L}{T_H}$$

c, b, a

$$(a) 1 - \frac{400}{800} = 1 - 0.5 = 0.5 \rightarrow 50\%$$

$$(b) 1 - \frac{600}{800} = 1 - 0.75 = 0.25 \rightarrow 25\%$$

$$(c) 1 - \frac{400}{600} = 1 - 0.66 = 0.34 \rightarrow 34\%$$



OPERATE BETWEEN  
OF (a) 400K AND 500K  
100 AND 600K. RANK  
TO THEIR THERMAL  
FIRST.

c, b, a

20%

→ 25%

4 → 34%

PH IMAGINE A CARNOT ENGINE THAT OPERATES BETWEEN  
THE TEMPERATURES  $T_H = 850K$  AND  $T_L = 300K$ .  
THE ENGINE PERFORMS 1200 J OF WORK EACH CYCLE.  
WHICH TAKES 0.25 SEC

(a) WHAT IS THE EFFICIENCY OF THIS ENGINE?

(b) WHAT IS THE AVERAGE POWER "P" OF THIS  
ENGINE?

(c) HOW MUCH ENERGY  $|Q_H|$  IS EXTRACTED  
AS HEAT FROM THE HIGH TEMPERATURE  
RESERVOIR EVERY CYCLE?

(d) HOW MUCH ENERGY  $|Q_L|$  IS DELIVERED  
AS HEAT TO THE LOW TEMPERATURE  
RESERVOIR EVERY CYCLE?

(e) BY HOW MUCH DOES THE ENTROPY  
OF WORKING SUBSTANCE CHANGE AS A  
RESULT OF THE ENERGY TRANSFERRED TO  
IT FROM THE HIGH TEMPERATURE  
RESERVOIR? (FROM IT TO THE LOW  
TEMPERATURE RESERVOIR?)

$$(a) e = 1 - \frac{T_L}{T_H} = 1 - \frac{300K}{850K}$$

$$(b) P = \frac{W}{t} = \frac{1200}{0.25} = 4800$$

$$(c) |Q_H| = \frac{W}{e} = \frac{1200}{0.25}$$

$$(d) |Q_L| = |Q_H| - W$$

$$= 1855 - 1200$$

$$= 655 J$$

$$(e) \Delta S_H = \frac{Q_H}{T_H}$$

$$\Delta S_L = \frac{Q_L}{T_L} =$$

NEGATIVE

TRANS

$$\Delta S = \Delta S_H +$$



ENGINE THAT OPERATES BETWEEN  
850K AND  $T_L = 300K$ .  
1200 J OF WORK EACH CYCLE.

EFFICIENCY OF THIS ENGINE?  
AVERAGE POWER "P" OF THIS

HOW MUCH  $|Q_H|$  IS EXTRACTED  
FROM THE HIGH TEMPERATURE  
RESERVOIR EACH CYCLE?

HOW MUCH  $|Q_L|$  IS DELIVERED  
TO THE LOW TEMPERATURE  
RESERVOIR EACH CYCLE?

WHAT IS THE ENTROPY  
CHANGE AS A  
RESULT OF ENERGY TRANSFERRED TO  
THE HIGH TEMPERATURE  
RESERVOIR FROM THE LOW  
TEMPERATURE RESERVOIR?

$$(a) e = 1 - \frac{T_L}{T_H} = 1 - \frac{300K}{850K} = 0.647 \Rightarrow 64.7\%$$

$$(b) P = \frac{W}{t} = \frac{1200}{0.25} = 4800W = 4.8kW$$

$$(c) |Q_H| = \frac{W}{e} = \frac{1200J}{0.647} = 1855J$$

$$(d) |Q_L| = |Q_H| - W \\ = 1855 - 1200 \\ = 655J$$

$$(e) \Delta S_H = \frac{Q_H}{T_H} = \frac{1855}{850} = 2.18 J/K$$

$$\Delta S_L = \frac{Q_L}{T_L} = \frac{-655}{300} = -2.18 J/K$$

NEGATIVE

TRANSFER

$$\Delta S = \Delta S_H + \Delta S_L = 2.18 + (-2.18)$$

$$= 0$$

PROBLEM: AN INVENTOR CLAIMS  
TO HAVE DESIGNED AN ENGINE  
THAT HAS AN EFFICIENCY OF 65%  
WHEN THE BOILING AND  
CONDENSING TEMPERATURES ARE  
100°C AND 20°C, RESPECTIVELY.  
IS THIS POSSIBLE?

$$e = 1 - \frac{T_L}{T_H} =$$

IM POSSIBLE



$$647 \Rightarrow 65\%$$

$$.8 \text{ kW}$$

$$= 1855 \text{ J}$$

$$= 2.18 \text{ J/K}$$

$$\frac{\Delta S}{\Delta t} = -2.18 \text{ J/K}$$

$$2.18 + (-2.18)$$

$$= 0$$

Pr AN INVENTOR CLAIMS TO HAVE CONSTRUCTED AN ENGINE THAT HAS AN EFFICIENCY OF 75% WHEN OPERATED BETWEEN THE BOILING AND FREEZING POINTS OF WATER, IS THIS POSSIBLE?

$$e = 1 - \frac{T_L}{T_H} = 1 - \frac{(0 + 273) \text{ K}}{(100 + 273) \text{ K}} = 1 - \frac{273}{373} \\ = 0.268 \approx 27\%$$

IM POSSIBLE



$$75 \Rightarrow 65\%$$

W

$$1855 \text{ J}$$

$$2.18 \text{ J/K}$$

$$-2.18 \text{ J/K}$$

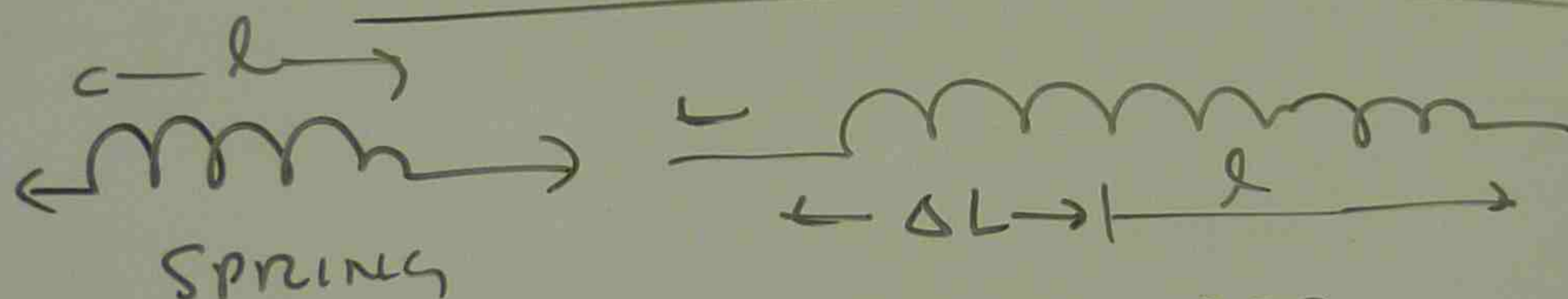
$$18 + (-2.18)$$

ph AN INVENTOR CLAIMS TO HAVE CONSTRUCTED AN ENGINE THAT HAS AN EFFICIENCY OF 75% WHEN OPERATED BETWEEN THE BOILING AND FREEZING POINTS OF WATER, IS THIS POSSIBLE?

$$e = 1 - \frac{T_L}{T_H} = 1 - \frac{(0 + 273) \text{ K}}{(100 + 273) \text{ K}} = 1 - \frac{273}{373} = 0.268 \approx 27\%$$

IM POSSIBLE

### HARMONIC MOTION AND WAVE



$$\text{ANGULAR VELOCITY } \omega = \sqrt{\frac{k}{m}}$$

$k$  = SPRING CONSTANT

$m$  = MASS

$$\text{FREQUENCY } f = \frac{\omega}{2\pi}$$

$$\text{PERIOD } T = \frac{1}{f}$$

ph THE BLOCK WHOSE MASS IS FASTENED TO A SPRING WITH CONSTANT  $k$  IS 65 N/m. THE BLOCK IS AT  $x = 11 \text{ cm}$  POSITION AT  $t = 0$  ON A SURFACE AND RELEASED AT  $t = 0$ .

(a) WHAT ARE ANGULAR FREQUENCY AND THE MOTION?

(b) WHAT IS THE AMPLITUDE?

(c) WHAT IS THE MAXIMUM VELOCITY OF THE OSCILLATING BLOCK WHEN IT IS AT THE EQUILIBRIUM POSITION?

(d) WHAT IS THE MAXIMUM ACCELERATION?

(e) WHAT IS THE PHASE OF THE MOTION?

(f) WHAT IS THE PERIOD?



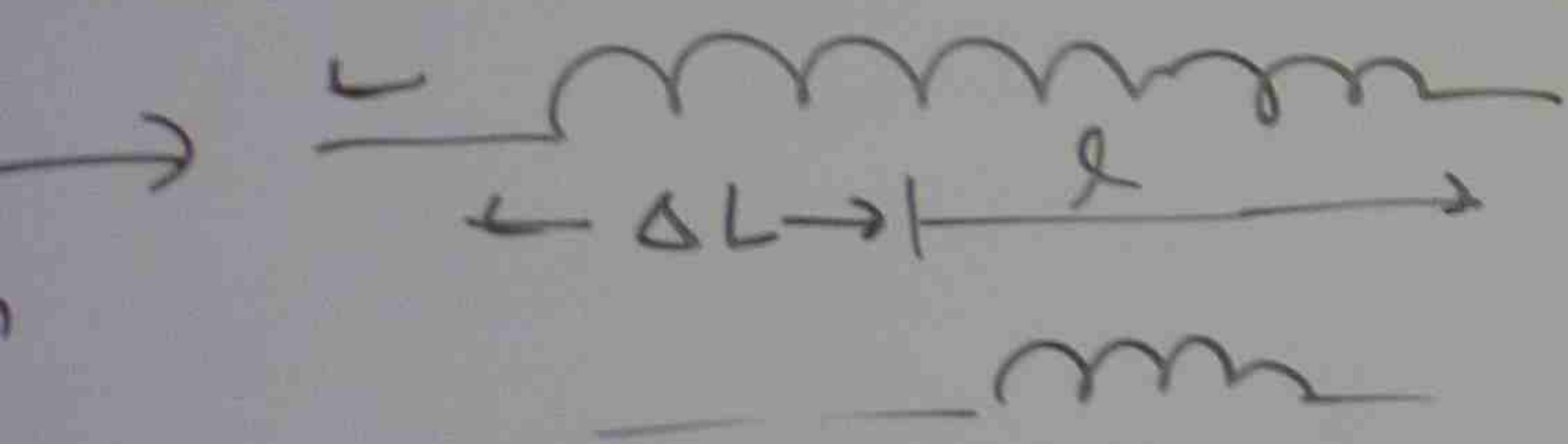
NEWTON CLAIMS TO HAVE CONSTRUCTED AN ENGINE  
AN EFFICIENCY OF 75% WHEN OPERATED BETWEEN  
G AND FREEZING POINTS OF WATER, IS THIS

$$\frac{T_L}{T_H} = 1 - \frac{(0 + 273) K}{(100 + 273) K} = 1 - \frac{273}{373}$$

$$= 0.268 \approx 27\%$$

IMPOSSIBLE

## HARMONIC MOTION AND WAVE



ANGULAR VELOCITY  $\omega = \sqrt{\frac{k}{m}}$

$k$  = SPRING  
CONSTANT

$m$  = MASS

FREQUENCY  $f = \frac{\omega}{2\pi}$

PERIOD  $T = \frac{1}{f}$

PH THE BLOCK WHOSE MASS " $m$ " IS 680 g  
IS FASTENED TO A SPRING WHOSE SPRING CONSTANT  
 $k$  IS 65 N/m. THE BLOCK IS PULLED A  
DISTANCE  $x = 11$  cm FROM ITS EQUILIBRIUM  
POSITION AT  $x = 0$  ON A FRICTIONLESS  
SURFACE AND RELEASED FROM REST AT  
 $t = 0$ .

- WHAT ARE ANGULAR FREQUENCY, THE  
FREQUENCY AND THE PERIOD OF RESULTING  
MOTION?
- WHAT IS THE AMPLITUDE OF OSCILLATION?
- WHAT IS THE MAXIMUM SPEED  $v_m$  OF  
THE OSCILLATING BLOCK AND WHERE IS THE  
BLOCK WHEN IT HAS THIS PERIOD?
- WHAT IS THE MAGNITUDE OF ACCELERATION?
- WHAT IS THE PHASE CONSTANT  $\phi$  FOR THE  
MOTION
- WHAT IS THE DISPLACEMENT FUNCTION?



$$(a) \omega = \sqrt{\frac{k}{m}} = \sqrt{\frac{69}{0.68}} = 9.78 \text{ rad/s}$$

$$f = \frac{\omega}{2\pi} = \frac{9.78}{2 \times 3.1416} = 1.6 \text{ Hz}$$

$$T = \frac{1}{f} = \frac{1}{1.6} = 0.64 \text{ sec} = 640 \text{ ms}$$

$$(b) X_m = 11 \text{ cm}$$

$$(c) V_m = \omega X_m = 9.78 \times \frac{11}{100} = 1.1 \text{ m/s}$$

$$(d) a_m = \omega^2 X_m = (9.78)^2 \times \frac{11}{100} = 11 \text{ m/s}^2$$

$$(e) \text{ AT TIME } t=0$$

$$X = X_m$$

$$\cos \phi = \frac{X}{X_m}$$

$$\cos \phi = \frac{X}{X_m}$$

$$\cos \phi = 1$$

$$\phi = 0$$

(f)

$$X(t) = X_m \cos(\omega t + \phi)$$

$$= (0.11 \text{ m}) \cos(9.78 t + \phi)$$

$$= 0.11 \cos(9.78 t + 0)$$

$$X(t) = 0.11 \cos 9.78 t$$

(a)

①  
③

pb At  $t=0$  THE DISPLACEMENT  $X(0)$  OF THE BLOCK IS  $-8.5 \text{ cm}$  THE BLOCK'S VELOCITY  $V(0)$  IS  $-0.92 \text{ m/s}$  AND ITS ACCELERATION  $a(0)$  IS  $47 \text{ m/s}^2$

(a) WHAT IS THE ANGULAR VELOCITY  $\omega$  OF THIS SYSTEM?

(b) WHAT ARE THE PHASE CONSTANT  $\phi$  AND AMPLITUDE  $X_m$ ?



$+ \phi)$   
 $8t + \phi)$   
 $t + 0)$   
 $t$

PLACEMENT

$S - 8.5 \text{ cm}$

$x(0)$  IS

ACCELERATION

ARE VELOCITY

$\text{cm?}$

PHASE CONSTANT

$x_m$

$$\begin{aligned} (a) \quad x(0) &= x_m \cos \phi & \text{--- (1)} \\ v(0) &= -\omega x_m \sin \phi & \text{--- (2)} \\ a(0) &= -\omega^2 x_m \cos \phi & \text{--- (3)} \end{aligned}$$

$$\frac{(1)}{(3)} \quad \frac{x(0)}{a(0)} = \frac{x_m \cos \phi}{-\omega^2 x_m \cos \phi}$$

$$\frac{x(0)}{a(0)} = -\frac{1}{\omega^2}$$

$$\omega^2 = -\frac{a(0)}{x(0)}$$

$$\omega = \sqrt{-\frac{a(0)}{x(0)}}$$

$$= \sqrt{-\frac{47}{(-8.5)}}$$

$$= 23.5 \text{ RAD/s}$$

(b)

$$\frac{(2)}{(1)} \Rightarrow \frac{v(0)}{x(0)} = \frac{-\omega x_m \sin \phi}{x_m \cos \phi}$$

$$\frac{v(0)}{x(0)} = -\omega \tan \phi$$

$$\tan \phi = -\frac{v(0)}{\omega x(0)} = \frac{-(-0.92)}{23.5 \times (-8.5)} = -0.461$$

$$\phi = -25^\circ \text{ (OR) } 155^\circ$$

$$x_m = \frac{x(0)}{\cos \phi}$$

$$\phi = -25^\circ \rightarrow x_m = \frac{-0.085}{\cos(-25^\circ)} = -0.094 \text{ m} = -9.4 \text{ cm}$$

$$\phi = 155^\circ \rightarrow x_m = \frac{0.085}{\cos 155^\circ} = 0.094 \text{ m} = 9.4 \text{ cm}$$

ENERGY

$U(t) =$

DAMP

mg

$x(t) =$

$\omega =$



## ENERGY IN SIMPLE HARMONIC MOTION

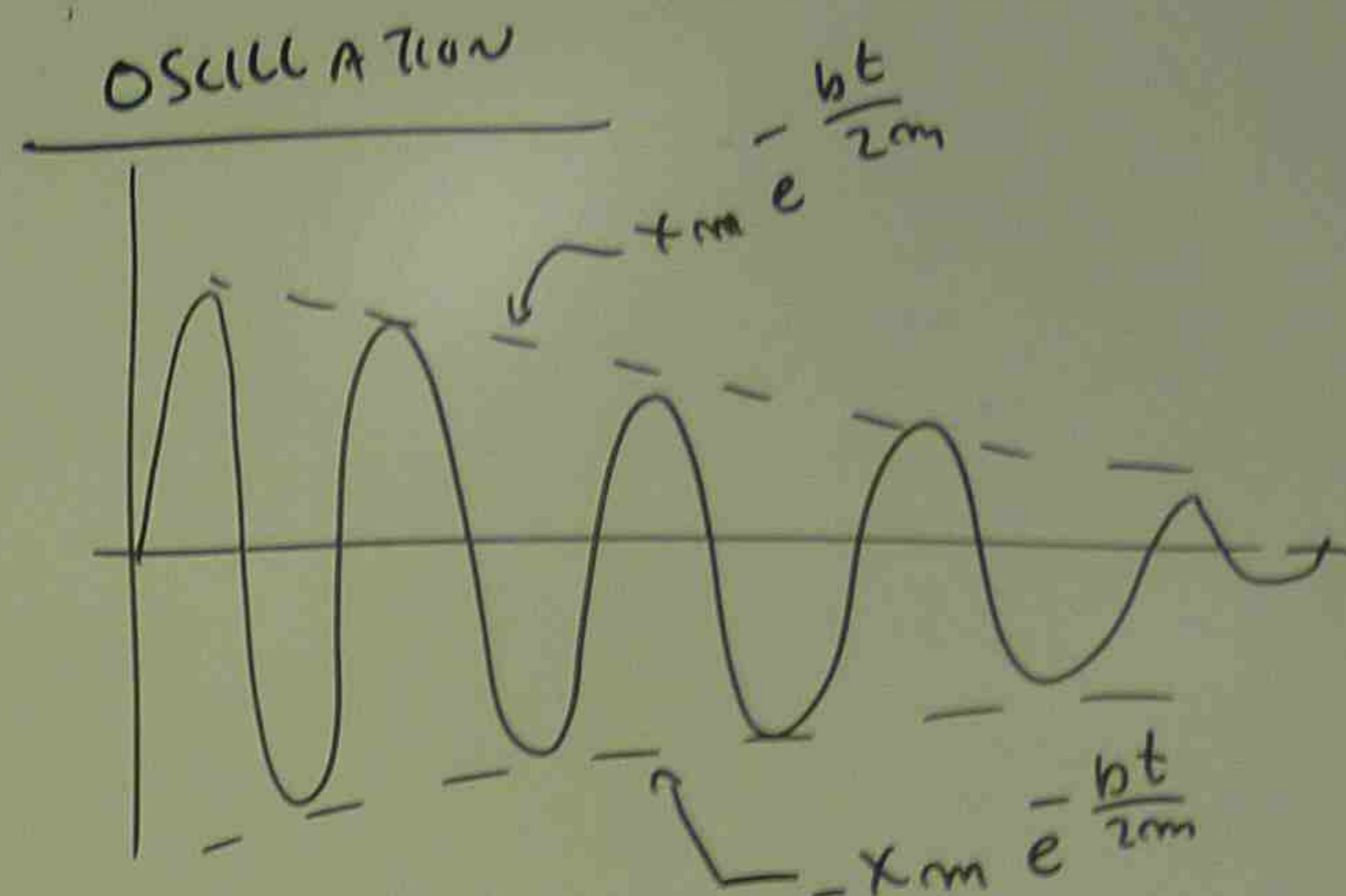
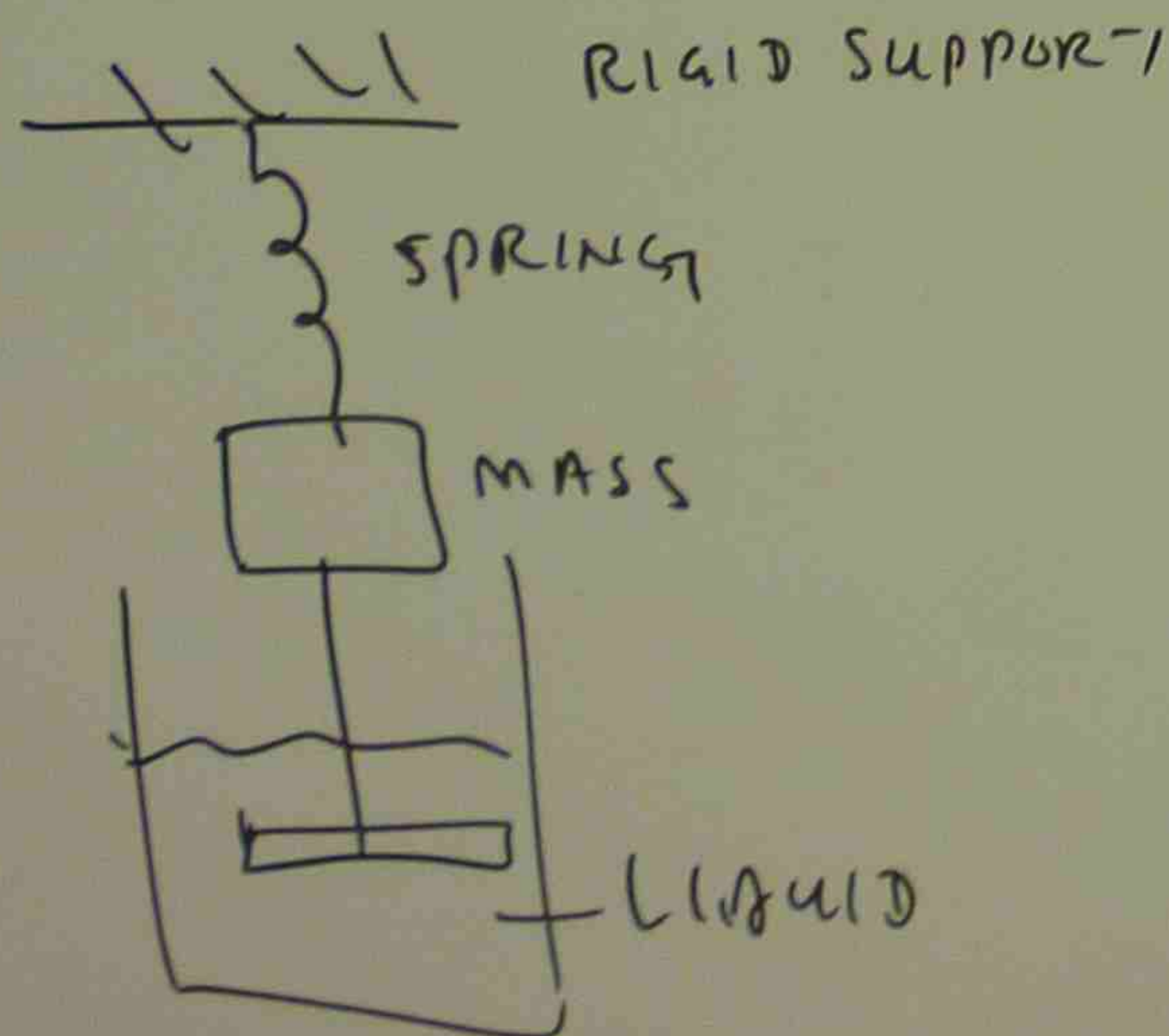
$$U(t) = \frac{1}{2} k x^2 = \frac{1}{2} k x_m^2 \cos^2(\omega t + \phi)$$

## DAMPED SIMPLE HARMONIC MOTION

$$m \frac{d^2 x}{dt^2} + b \frac{dx}{dt} + kx = 0$$

$$x(t) = x_m e^{-\frac{bt}{2m}} \cos(\omega' t + \phi)$$

$$\omega' = \sqrt{\frac{k}{m} - \frac{b^2}{4m^2}}$$



$$E(t) = \frac{1}{2} k x_m^2 e^{-\frac{bt}{m}}$$

PD FOR THE DAMPED OSCILLATOR  
 $m = 250 \text{ g}$ ,  $k = 89 \text{ N/m}$ ,  $b = 70 \text{ g/s}$

- WHAT IS THE PERIOD OF THE MOTION?
- HOW LONG DOES IT TAKE FOR THE AMPLITUDE OF THE DAMPED OSCILLATIONS TO DROP TO HALF ITS INITIAL VALUE?
- HOW LONG DOES IT TAKE FOR THE MECHANICAL ENERGY TO DROP TO ONE HALF ITS INITIAL VALUE?



$$(a) T = 2\pi \sqrt{\frac{m}{k}} = 2\pi \sqrt{\frac{0.25}{85}} = 0.34 \text{ sec}$$

$$(b) x(t) = x_m e^{-\frac{bt}{2m}}$$

$$\frac{1}{2} x_m = x_m e^{-\frac{bt}{2m}}$$

$$0.5 = e^{-\frac{bt}{2m}}$$

$$\ln 0.5 = -\frac{bt}{2m}$$

$$t = \frac{2m \ln 0.5}{-b}$$

$$= \frac{-2 \times 0.25 \ln 0.5}{0.070}$$

$$= 5 \text{ sec}$$

$$(c) \frac{1}{2} k x_m^2 e^{-\frac{bt}{m}} = \frac{1}{2} \left( \frac{1}{2} k x_m^2 \right)$$

$$e^{-\frac{bt}{m}} = 0.5$$

$$-\frac{bt}{m} = \ln 0.5$$

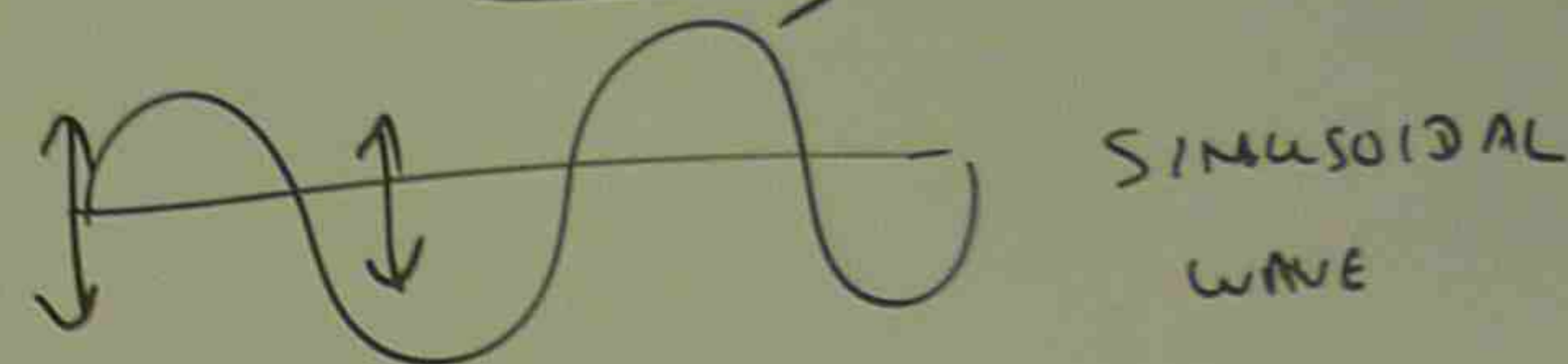
$$t = -\frac{\ln 0.5 \times m}{b} = \frac{-\ln 0.5 \times 0.25}{0.070} = 2.5 \text{ sec}$$

### Types of waves

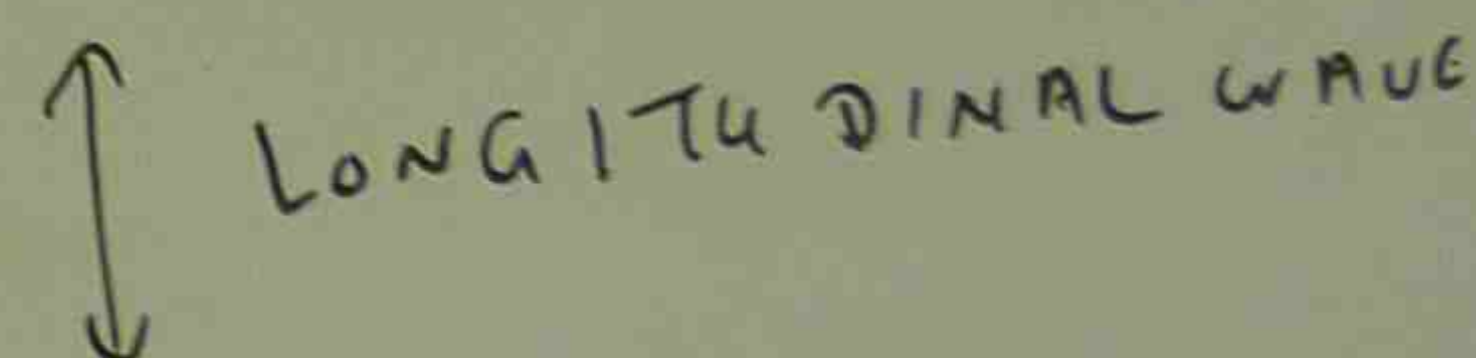
MECHANICAL WAVE  
ELECTROMAGNETIC WAVE  
MATTER WAVE



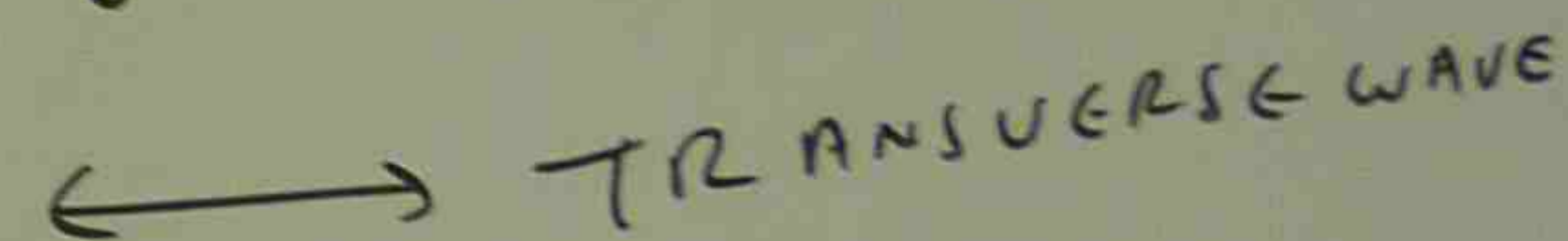
PULSE



SINUSOIDAL WAVE

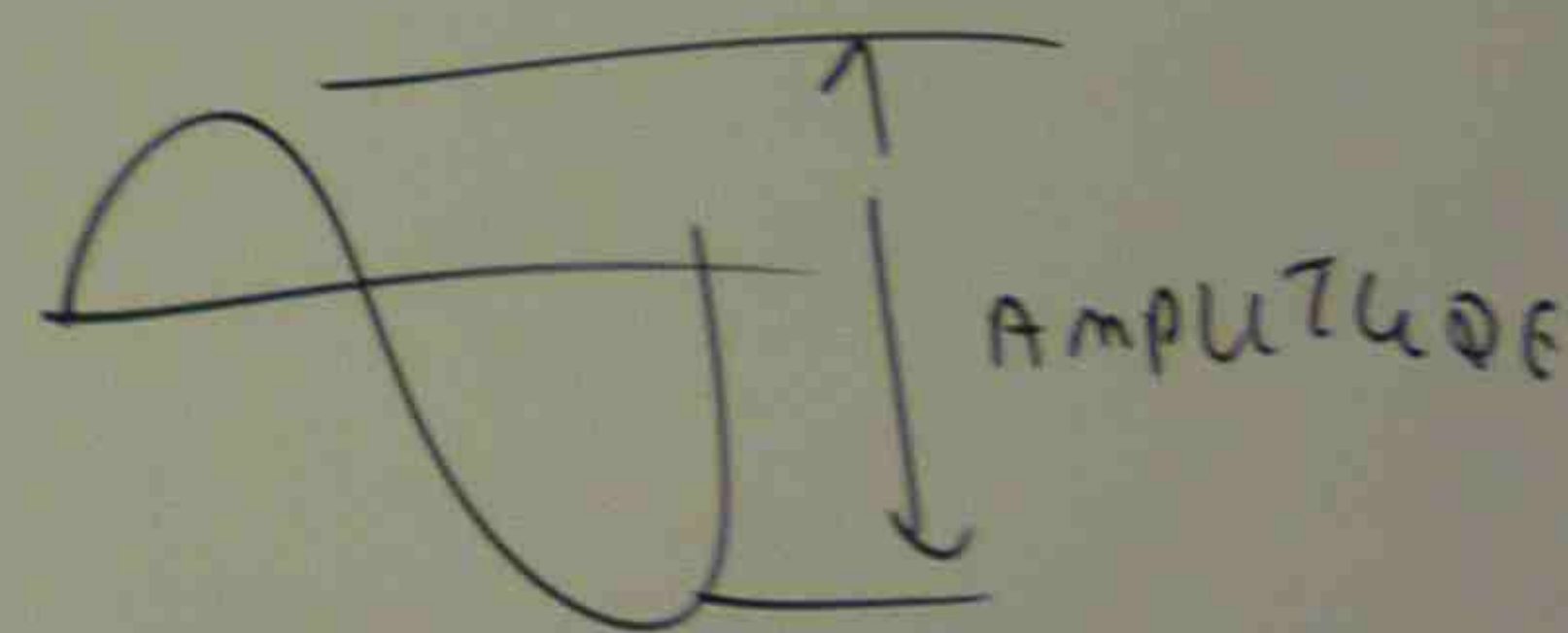
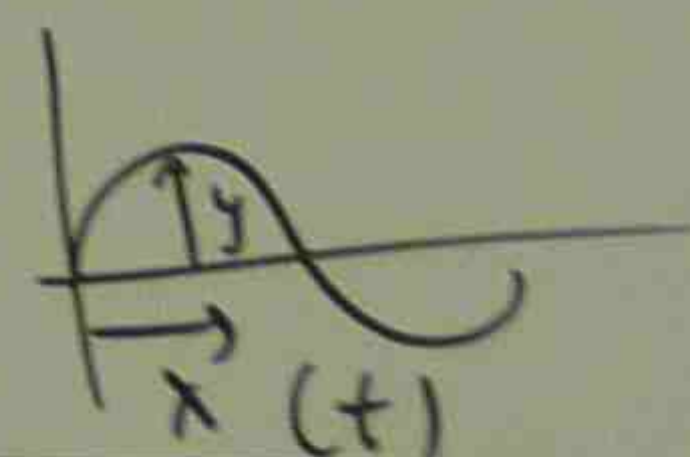


LONGITUDINAL WAVE

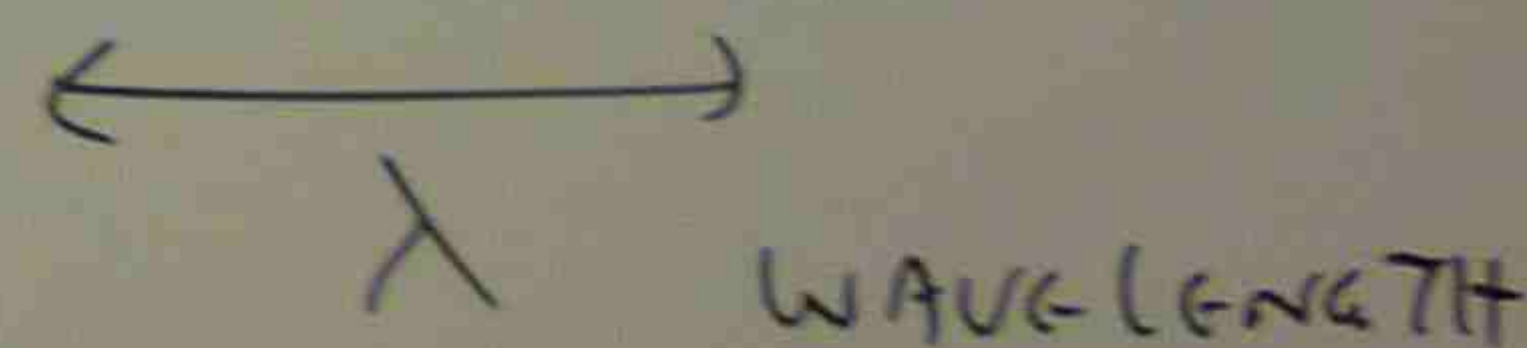


TRANSVERSE WAVE

$$y(x, t) = y_m \sin(kx - \omega t)$$



AMPLITUDE



WAVELENGTH

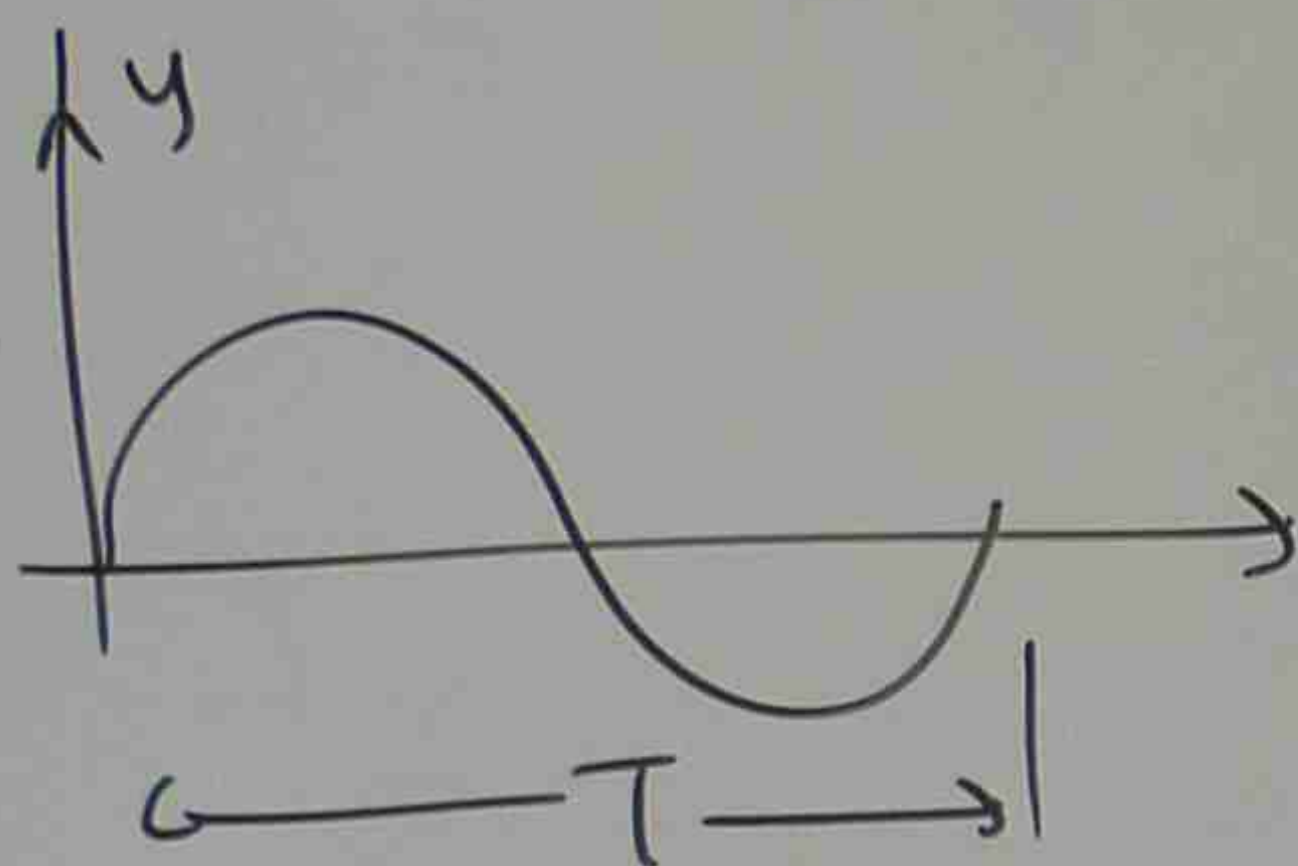


$$k = \text{ANGULAR WAVE NUMBER} = \frac{2\pi}{\lambda}$$

ANGULAR FREQUENCY =  $\omega = \frac{2\pi}{T}$

frequency  $f = \frac{1}{T} = \frac{\omega}{2\pi}$

$\lambda = \text{WAVE LENGTH}$ ,  $T = \text{PERIOD}$



## THE SPEED OF A TRAVELLING WAVE

$$K \propto L - \omega t = \text{CONSTANT}$$

$$v = \frac{\omega}{k} = \frac{\lambda}{T} = \lambda f$$

## LINEAR VELOCITY

(d)  $y = 8.0032 + \sin(72.1x - 2.72t)$

$$y = 0.0032 + \sin\left(72.1 \times \frac{22.5}{60} - 2.72 \times 18.9\right) \\ = 0.00192 \text{ m} \rightarrow 1.92 \text{ mm}$$

pn A WAVE TRAVELLING ALONG A STRING  
IS DESCRIBED BY

$$y(x,t) = 0.00327 \sin(72.1x - 2.72t)$$

(a) WHAT IS THE AMPLITUDE OF THIS WAVE?

(b) WHAT ARE THE WAVE LENGTH, PERIOD AND FREQUENCY OF THIS WAVE?

(c) WHAT IS THE VELOCITY OF THIS CURVE?

(d) WHAT IS THE DISPLACEMENT  $y$  AT  $x = 22.5 \text{ cm}$  AND  $t = 18.9 \text{ sec}$ ?

(a)  $x_{\text{cm}} = 0.00327 \text{ m} \rightarrow 3.27 \text{ mm}$   
 $\quad \quad \quad \downarrow \times 1000$

$$(b) \quad y(x, t) = 0.00327 \sin(72.1x - 2.72t)$$

$$y(x, t) = y_m \sin(kx - \omega t)$$

$$k = 72.1 \text{ RAD/m}, \omega = 2.72 \text{ RAD/s}$$

$$\lambda = \frac{2\pi}{k} = \frac{2 \times 3.1416}{72.1} = 0.0871 \text{ m} \rightarrow 8.71 \text{ cm}$$

$$T = \frac{2\pi}{\omega} = \frac{2 \times 3.1416}{7.72} = 2.31 \text{ sec}$$

$$f = \frac{1}{T} = \frac{1}{2.31} = 0.433 \text{ Hz}$$

$$(c) v = \frac{\omega}{k} = \frac{2.72 \text{ RAD/s}}{72.1 \text{ RAD/m}} = 0.0377 \text{ m/s}$$